beams: bending and shear stress

Pure Bending
- bending only
- no shear
- axial normal stresses from bending can be found in
  - homogeneous materials
  - plane of symmetry
  - follow Hooke’s law

Beam Bending
- Galileo
  - relationship between stress and depth^2
- can see
  - top squishing
  - bottom stretching

- what are the stress across the section?

Bending Moments
- sign convention:
- size of maximum internal moment will govern our design of the section
Normal Stresses

- geometric fit
  - plane sections remain plane
  - stress varies linearly

Neutral Axis

- stresses vary linearly
- zero stress occurs at the centroid
- neutral axis is line of centroids (n.a.)

Derivation of Stress from Strain

- pure bending = arc shape

\[ L = R \theta \]

\[ L_{\text{outside}} = (R + y)\theta \]

\[ \varepsilon = \frac{\delta}{L} = \frac{L_{\text{outside}} - L}{L} = \frac{(R + y)\theta - R \theta}{R \theta} = \frac{y}{R} \]

Derivation of Stress

- zero stress at n.a.

\[ f = E \varepsilon = \frac{E y}{R} \]

\[ f_{\text{max}} = \frac{E c}{R} \]

\[ f = \frac{y}{c} f_{\text{max}} \]
**Bending Moment**

- resultant moment from stresses = bending moment!

\[ M = \Sigma f_y \Delta A \]

**Bending Stress Relations**

\[ \frac{1}{R} = \frac{M}{EI} \]

\[ f_b = \frac{M}{I} \]

\[ S = \frac{I}{c} \]

- curvature
- general bending stress
- section modulus

\[ S_{\text{required}} \geq \frac{M}{F_b} \]

- maximum bending stress
- required section modulus for design

**Transverse Loading and Shear**

- perpendicular loading
- internal shear
- along with bending moment

**Bending vs. Shear in Design**

- bending stresses dominate

- shear stresses exist horizontally with shear

- no shear stresses with pure bending
Shear Stresses

• horizontal & vertical

Beam Stresses

• horizontal with bending

Equilibrium

• horizontal force V needed

\[ V_{\text{longitudinal}} = \frac{V \cdot Q}{I} \Delta x \]

• Q is a moment area
Moment of Area

- $Q$ is a moment area with respect to the n.a. of area above or below the horizontal
- $Q_{\text{max}}$ at $y=0$ (neutral axis)
- $q$ is shear flow: $q = \frac{V_{\text{longitudinal}}}{\Delta x} = \frac{V_T Q}{I}$

Shearing Stresses

- $f_v = \frac{V}{\Delta A} = \frac{V}{b \cdot \Delta x}$
- $f_{v-\text{ave}} = \frac{VQ}{Ib}$

- $f_v = 0$ on the top/bottom
- $b_{\text{min}}$ may not be with $Q_{\text{max}}$
- with $h/4 \geq b$, $f_{v-\text{max}} \leq 1.008 f_{v-\text{ave}}$

Rectangular Sections

- $I = \frac{bh^3}{12}$
- $Q = A\bar{y} = \frac{bh^2}{8}$
- $f_v = \frac{VQ}{Ib} = \frac{3V}{2A}$

- $f_{v-\text{max}}$ occurs at n.a.

Steel Beam Webs

- $W$ and $S$ sections
  - $b$ varies
  - stress in flange negligible
  - presume constant stress in web $f_{v-\text{max}} = \frac{3V}{2A} \approx \frac{V}{A_{\text{web}}}$
Shear Flow
- loads applied in plane of symmetry
- cut made perpendicular
\[ q = \frac{VQ}{I} \]

Shear Flow Quantity
- sketch from Q
\[ q = \frac{VQ}{I} \]

Connectors Resisting Shear
- plates with
  - nails
  - rivets
  - bolts
- splices
\[ \frac{V_{\text{longitudinal}}}{p} = \frac{VQ}{I} \]
\[ nF_{\text{connector}} \geq \frac{VQ_{\text{connected area}}}{I} \cdot p \]

Vertical Connectors
- isolate an area with vertical interfaces
Unsymmetrical Shear or Section

- member can bend and twist
  - not symmetric
  - shear not in that plane
- shear center
  - moments balance